

# Evaluation of <br> Methodology for LPG Fuel System Integrity Tank Test 

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16. Abstract

In response to a petition for rulemaking from General Motors Company to adopt new fuel system integrity requirements for liquefied petroleum gas (LPG) vehicles, the National Highway Traffic Safety Administration evaluated the feasibility of assessing fuel leakage from LPG vehicles after a crash test.

The primary challenge for evaluating the fuel system integrity of LPG-fueled vehicles is that LPG exists as both a liquid and a gas in the tank, which is not practical to exactly duplicate during a crash test. However, use of pressurized water and nitrogen was shown to be a reasonable simulation. Loss of "fuel" can be determined by post-crash test measurements of both pressure drop in the tank and volume of liquid lost. Depending on the number of leaks and their locations, a measured post-test pressure drop may be due to a gaseous leak, a liquid leak (pressure drop due to increased gas volume in tank from loss of liquid), or both. Also, the pressure drop due to only a marginal loss of liquid ( 805 g of water) is relatively small compared to the total tank pressure, particularly for larger LPG tanks.

Results of this project showed promise for using pressure drop and liquid fluid leakage measurements to evaluate the fuel system integrity of LPG-fueled vehicles during crash tests. This study showed that pressure transducers and data acquisition systems are available that can accurately measure the relatively small pressure drop associated with a marginal failure condition, but care must be taken to account for ambient pressure and temperature changes.

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## EXECUTIVE SUMMARY

In response to a petition for rulemaking from General Motors Company ${ }^{1}$ to adopt new fuel system integrity requirements for liquefied petroleum gas (LPG) ${ }^{2}$ fueled vehicles, the National Highway Traffic Safety Administration evaluated the feasibility of assessing fuel leakage from LPG vehicles after a crash test.

Specifically, the agency examined allowable levels of LPG spillage and pressure drop that was modeled after the theoretical derivation of the allowable compressed natural gas leakage specified in Federal Motor Vehicle Safety Standard (FMVSS) No. 303, "Fuel system integrity of compressed natural gas (CNG) vehicles," and that utilized the equivalent energy content of the allowable gasoline spillage requirement specified in FMVSS No. 301, "Fuel system integrity." Leakage would then be evaluated by examining the calculated amount of liquid propane that escapes from the tank in an equivalent period to gasoline and/or the amount of propane gas escaping from the tank during that same period. The allowable loss of LPG would be equivalent, based on the energy content of the fuel, to the allowable leakage of gasoline in FMVSS 301.

The primary challenge for evaluating the fuel system integrity of LPG-fueled vehicles is that LPG exists as both a liquid and a gas in the tank. However, use of pressurized water and nitrogen was shown to be a reasonable simulation. Loss of 'fuel' can be determined by postcrash test measurements of both pressure drop in the tank and the volume of liquid lost. Depending on the number of leaks and their locations, a measured post-test pressure drop may be due to a gaseous leak, a liquid leak (pressure drop due to increased gas volume in tank from loss of liquid), or both.

Results of this testing showed promise for using pressure drop and liquid fluid leakage measurements to evaluate the fuel system integrity of LPG-fueled vehicles during crash tests. This study showed that pressure transducers and data acquisition systems are available that can accurately measure these pressure drops, but care must be taken to account for ambient pressure and temperature changes.

## 1.0 - INTRODUCTION

In response to a petition for rulemaking from GM to adopt new fuel system integrity requirements for LPG fueled vehicles, NHTSA evaluated the feasibility of assessing fuel leakage from LPG vehicles after a crash test. Both FMVSS No. 301 for liquid-fueled vehicles and FMVSS No. 303 for CNG fueled vehicles require fuel system integrity in frontal, rear, and lateral barrier crashes for vehicles with a gross vehicle weight rating (GVWR) of 10,000 lb or less and in contoured barrier crash tests for school buses with a GVWR over $10,000 \mathrm{lb}$.

Propane is a gas at normal atmospheric pressure and temperature. ${ }^{3}$ In a vehicle fuel tank, the propane would be in both a liquid and gas phase when under pressure. ${ }^{4}$ Because of this, potential fuel leakage cannot be determined by direct measurement of the amount of fuel escaping from the vehicle as done for FMVSS No. 301. ${ }^{5}$

[^0]Leakage would then be evaluated by examining the calculated amount of liquid propane that escapes from the tank in an equivalent period to gasoline and/or the amount of propane gas escaping from the tank during that same period. The allowable loss of LPG would be equivalent, based on the energy content of the fuel, to the allowable leakage of gasoline in FMVSS No. 301. Since the LPG tank contains both a liquid and a gas, both leak rates have to be examined. Additionally, since the vehicle may undergo rotation in a fixture, similar to FMVSS No. 301, methods to evaluate and combine both the liquid and gas leakage rates may need to be considered.

NHTSA conducted this project at its Vehicle Research and Test Center (VRTC) to support the agency's future consideration of fuel systems integrity of LPG vehicles, by assessing the practical considerations involved in measuring LPG leakage after a crash test. The pressure drop evaluated during these tests represented only a marginal failure condition, such as a leak due to minor damage to fuel lines, since determination of non-compliance for a large leak is more evident.

This program consisted of the following:

1. Laboratory tests to evaluate the validity of measuring pressure drop as a result of leakage of an equivalent volume of water for small LPG tanks. This also involved evaluating the accuracy of commercially available instrumentation for measuring the relatively small pressure drop associated with a marginal failure condition.
2. Laboratory tests to evaluate the validity of measuring pressure drop as a result of leakage of an equivalent volume of water for large LPG tanks, where the change in pressure would be less than in a small tank, for the same volume of fuel loss.
3. Laboratory tests to evaluate the validity of measuring pressure drop as a result of leakage of a volume of gaseous propane through the identical size orifice that gave the maximum acceptable leak rate for water.

This report contains the results and findings from this study.

## 2.0 - BACKGROUND

The behavior of a real gas, at sufficiently low density, can be closely approximated by the ideal gas equation of state:

$$
P V=n R T
$$

Where:
$\mathrm{P}=$ absolute pressure Newton $/$ meters $^{2}\left(\mathrm{~N} / \mathrm{m}^{2}\right)$
$\mathrm{V}=$ volume ( $\mathrm{m}^{3}$ )
$\mathrm{n}=$ chemical amount (moles)
$R=$ universal gas constant Joules/mole degree Kelvin ${ }^{6}\left(\mathrm{~J} / \mathrm{mole}^{*} \mathrm{~K}\right)$
$\mathrm{T}=$ temperature (K)
A slightly modified version of this equation adds a correction factor to account for the incompressibility of real gases. For nitrogen (the gas used for testing), which closely approximates an ideal gas over a large range of temperatures and pressures, this equation is as follows:

$$
P V=Z n R T
$$

Where:
$\mathrm{Z}=$ Correction Factor specific gas; for Nitrogen: Z is dimensionless with value near $1 .{ }^{7}$
A list of acronyms used in this report is provided in Appendix A1, and a list of references consulted for this research work is provided in Appendix A2.

For a constant volume system, this indicates that pressure drop is directly proportional to the temperature of the gas. For a system with both liquid and gas in a pressurized vessel, loss of liquid will result in an increase in gas volume along with a decrease in pressure. Alternatively, leakage can result in a loss of gas ( n , moles of nitrogen). This in turn will affect the pressure and temperature equilibrium states of the gas. Gas temperature is also affected by the ambient temperature, and thus, gas pressure.

A series of bench tests was performed at VRTC to study the practical effects of gas loss on pressure and temperature in an LPG fuel system. The gas used in all of these tests was industrial (welding) grade nitrogen gas. The results and conclusions presented here are based on the results using nitrogen as the working gas. The behavior of propane in the gaseous state is generally considered to be a reasonable approximation of an ideal gas under the ranges of temperature and pressure discussed in these tests.

[^1]
## 3.0 - TEST PROCEDURE

The test procedure evaluated in this study was modeled after Canada's test procedure for testing the fuel system integrity of LPG vehicles. This involves filling the cylinder with water equivalent to that of the weight of LPG, pressurizing the remainder of the tank with nitrogen gas, and then conducting the crash test.

## 3.1 - Test Procedures - General

For the laboratory bench tests in this study, rather than impact the cylinders to induce damage and create a leak that is uncontrolled and unpredictable, the cylinders were retrofitted with regulators so that a controlled leak could be achieved (thus allowing an unlimited number of tests with the same cylinder).

The cylinders came with the necessary valves (fill, service, relief, etc.) as well as gauges (pressure, float, and fixed liquid level). Two commercial cylinders, a small 26.2 gallon ( 99.2 liters ( L )) and a large 41.4 gallon ( 156.7 L ) cylinder were fitted with a flow regulator that could control the amount of liquid output. Specifications of the two tanks used in the testing are provided in Appendix D. The cylinders were filled to 40 percent of capacity with water (weight equivalent substitute for $80 \%$ LPG fill). ${ }^{89}$ Using the fill valve, the remainder of the cylinder was pressurized with nitrogen to a gauge pressure of 292 pounds per square inch ( psig ), which was 20 pounds per square inch ( psi ) below the 312 psig maximum operating pressure of the both tanks as specified by the manufacturer.

The maximum allowable leak of liquid propane was based on the equivalent energy of the maximum allowable gasoline leak specified in FMVSS No. 301 (details provided in Appendix C1). The maximum allowable gasoline leak is 870 grams ( g ) in 30 minutes after a crash test. The equivalent energy maximum allowable LPG leak is 805 g . ${ }^{10}$

Tests were then conducted to verify the following:

1. That a leak in the liquid (water) portion of each tank through the flow regulator (at various rates up to 805 grams ${ }^{11} 12$ total in 30 minutes) will or will not lead to a detectable pressure drop of nitrogen.
2. That a leak in the gaseous portion of the tank from pressure measurement sensors (at various rates) is detectable. This was then compared to a historical reference of a 5 percent ${ }^{13}$ cumulative potential measurement error as measured by the pressure gauges.
3. That the total pressure drop is less than $15.5 \mathrm{psi}^{14}$ over 60 minutes.

[^2]4. The accuracy at which current pressure transducers are capable of measuring at the targeted normal working pressure range (200-300 psi).

These tests were done at room temperature. A reference pressure transducer, thermocouples (tank and ambient temperature), a pressure transducer for ambient air pressure, and a digital scale to measure mass of water drained from the cylinder were used. Because the volume of liquid water drained was known (i.e. the volume for 805 g of water is 805 cubic centimeters (cc)), this became an exercise in verifying Boyle's law with initial pressures and all volumes known.

$$
P_{i} V_{i}=Z_{N 2} n R T_{i}
$$

and

$$
P_{f} V_{f}=Z_{N 2} n R T_{f}
$$

then

$$
P_{f}-P_{i}=\left(Z_{N 2} n R T_{f}\right) / V_{f^{-}}\left(Z_{N 2} n R T_{i}\right) / V_{i}
$$

Assuming no change in ambient temperature or ambient pressure we have

$$
P_{i} V_{i}=P_{f} V_{f}
$$

Where $V_{i}=0.6 V \mathrm{cc}, V_{f}=0.6 V+805 \mathrm{cc}, P_{i}=P_{\max }=P_{\text {start }}$

$$
\begin{gathered}
\Delta P=P_{i}-P_{f}=P_{i}\left(1-\frac{V_{i}}{V_{f}}\right) \\
\Delta P=\left(P_{\max }-P_{\text {start }}\right)^{*}[1 /\{1+(0.6 \mathrm{~V} / 805)\}]
\end{gathered}
$$

Where:

```
\(P_{\max }=\) max operating pressure \({ }^{15}\) (psig)
\(P_{\text {start }}=\) test start operating pressure reduction (generally 20 psig\()^{16}\)
\(V=\) volume of tank
\(\Delta P=\) change in tank pressure after fluid loss \({ }^{17}\)
\(Z_{N 2}=\) compressibility factor nitrogen
\(n=\) number of moles
```

There was a larger pressure drop initially because of temperature drop (due to expansion of the gas) when the liquid is being drained, which will partially recover as the temperature recovers. For a loss of 805 g of water, the theoretical change (at normal temperature and

[^3]pressure, NTP) ${ }^{18}$ in pressure will be -4.1 psia for the small 26.2 gallon tank and -2.6 psia for the large 41.4 gallon tank (once the pressure and temperature have stabilized to ambient conditions).

Additional testing and evaluations notes:

1. Collect temperature and pressure data for different flow rates of liquid water drainage over 30 minutes.
2. Evaluate the pressure drop and energy loss of gas flowing through an orifice size that had been adjusted to allow 805 g of fluid to "leak" in 30 minutes.
[^4]
## 3.2 - Test Setup and System/Component Description ${ }^{19}$

The test setup included a high pressure nitrogen tank, multiple pressure transducers, a tank thermocouple to measure nitrogen temperature (at the center of the circumference inside the tank), and a digital scale to measure the mass of water being evacuated from the device under test (DUT). The test setups for the 26.2 gallon and 41.4 gallon LPG tanks are shown in Figures 1 and 2, respectively. ${ }^{20}$ The tanks each have a "FILL" ${ }^{21}$ port that opens to the top of the tank, a "LIQUID" port which opens at the bottom of the tank, and a "RELIEF" port where pressure transducers were attached.


Figure 1 - Typical 26.2 Gallon LPG Automotive Fuel Tank with Test Components

[^5]

Figure 2-41.4 Gallon LPG Tank Test Setup

A needle valve and a shutoff valve were attached to the liquid port on each tank. The fill port also had a shutoff valve and funnel for reintroducing liquid back into the tank after each test. The data acquisition system consisted of dedicated analog signal conditioners connected to a high speed multiplexer, and the data were processed with VRTC in-house software (see Figure 3). ${ }^{22}$ Additionally, an ambient pressure transducer to measure atmospheric pressure and an ambient temperature thermocouple were used to evaluate temperature in the building that might affect results. ${ }^{23}$ Details of instrumentation in these tests are provided in Appendix A3. ${ }^{19}$

[^6]

Figure 3 - Data Acquisition System with Ambient Pressure and Temperature Transducers

Tests were typically run with the specified leak rate over a 30 -minute or 60 -minute period, and data was collected over a 120 -minute period to evaluate pressure and temperature stabilization and recovery. Following each test, the tank was depressurized and the water "leaked" from the tank was simply reintroduced into the tank. Then for the next test, the tank was repressurized to the desired level and allowed to stabilized, with additional minor repressurizations as needed.

## 3.3 - Test Procedure - Liquid Leak Testing - Details

1. As mentioned above, the LPG tanks were filled with water to 40 percent of their capacity. The weight of the water added to the small tank was 88.45 lb , and 139.77 lb of water was added to the large tank. We allowed an overnight soak so that the tank skin and water could reach approximate ambient temperature. ${ }^{24}$
2. The LPG tank was then pressurized to approximately 292 psi (actual pressure was temperature corrected) through at least two pressurizations, allowing for pressure and temperature stabilizations after each fill. ${ }^{25}$
3. The LPG tanks were fitted with a highly accurate pressure transducer ${ }^{26}$ and a pressure gauge to monitor tank pressure. Tank temperature (in the gaseous portion of the tank), ambient temperature, and ambient pressure were also monitored on a digital data acquisition system.

[^7]4. Water flow was controlled with a needle valve, and flow rate was measured with a scale to regulate the exact mass of water leaving the tank (at the conclusion of each test, the water was reentered into the tank to maintain the exact required mass of the water) (see Figures 4 through 8).
5. The desired flow rate in grams of water was calculated, and the time for this flow for each 100 g of water was tabulated. Then a stopwatch was used to record the time for each 100 g increment of water to flow out of the tank. In this way, the flow could be adjusted to get a very precise mass flow rate to match the required output.
6. After the required 30 -minute period of flow monitoring was done, the LPG tank pressures, as well as other parameters, were monitored for a total of 120 minutes to evaluate the drift and the time to stabilize temperature and pressure from the initial liquid loss.

[^8]


Figure 6 - Pressure Transducers - Close Up


Figure 7 - Ambient Pressure Transducer
Figure 8 - Ambient Temperature Gauge

## 3.4 - Test Procedure - Gaseous Leak Testing

Since an LPG fuel system integrity test could also include rotation of the post-crash test vehicle, similar to FMVSS No. 301 testing of a gasoline powered vehicle, we evaluated the effects of an identical size leak with gas escaping instead of liquid escaping. To do this, we set the needle valve that had controlled the water flow rate during the liquid escape tests to the exact orifice size which allowed 805 g of water to leak in 30 minutes. This needle valve setting was secured and the valve (red arrow Figure 10) was moved from the liquid fill connection to a port that connected to the gas portion of the tank (see also Figure 9). The pressure drop through this size orifice over a 30 -minute period was monitored and recorded.


Figure 9 - Overall Test Setup
Figure 10 - Needle Valve on Gas Portion of Tank

## 4.0 - RESULTS

This section contains the theoretical calculations and test results.

## 4.1 - Small Tank Leak Testing Results

### 4.11 - Theoretical Calculations

Theoretical calculations for the 26.2 gallon ( 99.2 L ) tank are shown in Table 1. These calculations are for a leak of exactly 805 g of water, at normal temperature and pressure (NTP), assuming no temperature change in the tank or ambient barometric pressure change during the event.

Table 1 - Theoretical Calculations for Drain of 805 g Water for 30 min - $\mathbf{2 6 . 2}$ Gallon Tank

| RUN No. |  | Tank <br> Total <br> Volume | Water <br> Volume <br> in Tank | Nitrogen Volume in Tank | Weight Water Drain | Change Water Volume | Tank Pres. | Ambient Pres. | Tank Pres. | Ambient Temp | Tank Temp (gas) | $\begin{gathered} \text { Pressure } \\ 68^{\circ} \mathrm{F} \\ 14.7 \mathrm{psia} \end{gathered}$ | Actual $\Delta$ Tank Pres. @NTP |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\mathrm{a} / \mathrm{n}$ | min | L | L | L | g | L | psig | psia | psia | ${ }^{\circ} \mathrm{F}$ | ${ }^{\circ} \mathrm{F}$ | psia | psia |
| TEST CALC | NA | 99.18 | 40.12 | 59.06 | 0.0 | 0.000 | 292.00 | 14.70 | 306.70 | 68.0 | 68.0 | 306.70 |  |
| ALL NTP | NA | 99.18 | 39.32 | 59.86 | 805.0 | -0.805 | 287.88 | 14.70 | 302.57 | 68.0 | 68.0 | 302.57 | -4.12 |

Providing the equations for the above we have:

Calculating the pressure loss for a given volume of water drained
We have the $\mathrm{N}_{2}$ volume of the tank as

$$
V_{(N 2 i)}=V_{(\text {tank })}-V_{(H 2 O i)}
$$

The final N 2 tank volume after water drainage is

$$
\begin{aligned}
& V_{(N 2 f)}=V_{(N 2 i)}+V_{(H 2 O f)} \\
& \Delta V_{(N 2)}=V_{(N 2 f)}-V_{(N 2 i)}
\end{aligned}
$$

Measuring pressure change and converting to absolute pressure we have:

$$
\begin{gathered}
P_{t(N 2 i)} \text { absolute }=P_{t(N 2 i)} \text { gauge }+P_{a(i)} \text { ambient air pressure }(\text { raw })^{27} \\
P_{t(N 2 f)} \text { absolute }=P_{t(N 2 f)} \text { gauge }+P_{a(f)} \text { ambient air pressure }(\mathrm{raw})
\end{gathered}
$$

Correcting temperature to absolute temperature we have:

$$
\begin{aligned}
& \left.T_{(N 2 i)} K=\left(T_{(N 2 i)}{ }^{\circ} F-32\right) *(5 / 9)+273.15\right) \\
& \left.T_{(N 2 f)} K=\left(T_{(N 2 f)}{ }^{\circ} F-32\right) *(5 / 9)+273.15\right)
\end{aligned}
$$

Calculating pressure correcting for temperature change at normal temperature (NT) we have:

$$
\begin{aligned}
& P_{t(N 2 i)} \text { at } N T=P_{t(N 2 i)} \text { absolute } *\left(293.15 \mathrm{~K} /+T_{(N 2 i)} K\right) \\
& P_{t(N 2 f)} \text { at } N T=P_{t(N 2 f)} \text { absolute } *\left(293.15 \mathrm{~K} /+T_{(N 2 f)} K\right)
\end{aligned}
$$

Finally we have

$$
\Delta P_{t(N 2)}=P_{t(N 2 f)} \text { at } N T-P_{t(N 2 i)} \text { at } N T
$$

Where
$V_{(N 2 i)}$ is $\mathrm{N}_{2}$ volume tank initial
$V_{(N 2 f)}$ is $\mathrm{N}_{2}$ volume tank final
$\left.V_{(H 2 \mathrm{O}}{ }^{i}\right)$ is water volume tank initial
$V_{\left(H 2 O_{f)}\right.}$ is water volume tank final
$V_{(\text {tank) }}$ is total volume of tank
$P_{t(N 2 i)}$ is $\mathrm{N}_{2}$ pressure tank initial
$P_{t(N 2 f)}$ is $\mathrm{N}_{2}$ pressure tank final
$P_{a(i)}$ is atmospheric pressure (raw) initial
$P_{a(f)}$ is atmospheric pressure (raw) final
$T_{(N 2 i)}$ is $\mathrm{N}_{2}$ temperature initial
$T_{(N 2 f)}$ is $\mathrm{N}_{2}$ temperature final
$\Delta P_{(N 2)}$ is change in tank pressure

[^9]
### 4.12 - Drain of 805 g Water in 30 min - One-Hour Test

Run 1140 was a typical test with the tank filled with water to 40 percent of its capacity and the pressure in the tank at $292 \mathrm{psig}(+/-2 \%)$, as corrected to normal temperature. ${ }^{28}$ This run produced the following results (see detailed final results in Table 2, summary data in Table 3, and pressure and temperature graphed as a function of time in Figure 11). ${ }^{29} 30$

Table 2 - Run 1140-26.2 Gallon Tank - Drain 805 g Water in 30 min - One-Hour Test ${ }^{31}$

| $\begin{aligned} & \text { RUN } \\ & \text { No. } \end{aligned}$ |  | Tank Total Volume | Water Volume in Tank | $\mathrm{N}_{2}$ Volume in Tank | Weight Water Drain | Change Water Volume | Tank Pres. | Ambient Pres. | Tank Pres. | Ambient Temp | Tank Temp (gas) | Pressure $68{ }^{\circ} \mathrm{F}$ 14.7 psia ambient | Actual $\Delta$ Tank Pres. @NTP |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| min | min | L | L | L | g | L | psig | psia | psia | ${ }^{\circ} \mathrm{F}$ | ${ }^{\circ} \mathrm{F}$ | psia | psia |
| $\begin{aligned} & \text { RUN } \\ & 1140 \end{aligned}$ | 0 | 99.2 | 40.1 | 59.1 | 0.0 | 0.000 | 296.6 | 14.2 | 310.8 | 74.0 | 76.1 | 306.0 |  |
| 30 min DRAIN @ 30 min | 30 | 99.2 | 39.3 | 59.9 | 805.2 | -0.805 | 292.2 | 14.2 | 306.4 | 75.0 | 75.8 | 301.9 | -4.1 |
| 30 min @ 60 min | 60 | 99.2 | 39.3 | 59.9 | 805.2 | -0.805 | 292.4 | 14.2 | 306.6 | 75.0 | 76.0 | 302.0 | -4.1 |

[^10]Table 3 - Test 1140 - 26.2 Gallon Tank, Drain of 805 g Water in 30 min , One-Hour Test - Summary ${ }^{32}$

| Time | Ambient <br> Pressure | Tank <br> Pressure | Tank <br> Temperature | Ambient <br> Temperature | Cumulative <br> Water <br> Leakage |
| :---: | :---: | :---: | :---: | :---: | :---: |
| min | psia (raw) | Psig | ${ }^{\circ} \mathrm{F}$ | ${ }^{\circ} \mathrm{F}$ | g |
| 0 | 14.2 | 296.6 | 76.1 | 75.3 | 0 |
| 10 | 14.2 | 295.1 | 76.0 | 75.3 | 262 |
| 20 | 14.2 | 293.7 | 75.9 | 75.5 | 536 |
| 30 | 14.2 | 292.2 | 75.8 | 75.7 | 805 |
| 40 | 14.2 | 292.4 | 76.0 | 75.9 | 805 |
| 50 | 14.2 | 292.4 | 76.0 | 75.9 | 805 |
| 60 | 14.2 | 292.4 | 76.0 | 76.4 | 805 |



Figure 11 - Test 1140 - 26.2 Gallon Tank, Drain of 805 g Water in 30 min - Pressure, Temperature $=\mathbf{f}(\mathbf{t i m e})^{33}$

[^11]
### 4.13 - Drain 805 g of Water in 30 min - Two-Hour Test

Another run with identical flow rate with a longer stabilization time was performed to compare results. Here the tank again was filled with water to 40 percent of its capacity and the pressure in the tank set at 292 psig ( $+/-2 \%$ ), as corrected to normal temperature. After the $30-$ minute drain, we continued to record tank pressure and temperature for a 2 - hour period to evaluate how the pressure stabilized over this longer period of time. This run produced the following results (see detailed final results in Table 4, and summary data in Table 5, and pressure and temperature graphed as a function of time in Figure 12).

Table 4 - Run 1189-26.2 Gallon Tank - Drain of 805 g Water in 30 min - Two-Hour Test

| RUN No. |  | Tank Total Volume | Water Volume in Tank | $\mathrm{N}_{2}$ Volume in Tank | Weight Water Drain | Change Water Volume | Tank Pres. | Ambient Pres. | Tank Pres. | Ambient Temp | Tank Temp (gas) | Pressure $68^{\circ} \mathrm{F}$ 14.7 psia ambient | Actual <br> $\Delta$ Tank Pres. @NT |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| min | min | L | L | L | g | L | psig | psia | psia | ${ }^{\circ} \mathrm{F}$ | ${ }^{\circ} \mathrm{F}$ | psia | psia |
| $\begin{aligned} & \text { RUN } \\ & 1189 \end{aligned}$ | 0 | 99.2 | 40.1 | 59.1 | 0.0 | 0.000 | 296.0 | 14.1 | 310.1 | 74.0 | 75.9 | 305.6 |  |
| 30 min DRAIN@ 30 min | 30 | 99.2 | 39.3 | 59.9 | 804.3 | -0.804 | 291.7 | 14.1 | 305.8 | 75.0 | 75.5 | 301.5 | -4.1 |
| 30 min DRAIN@ 60 min | 60 | 99.2 | 39.3 | 59.9 | 804.3 | -0.804 | 291.8 | 14.1 | 305.9 | 75.0 | 75.6 | 301.6 | -4.0 |
| 30 min DRAIN@ 120 min | 120 | 99.2 | 39.3 | 59.9 | 804.3 | -0.804 | 291.7 | 14.1 | 305.8 | 75.0 | 75.5 | 301.5 | -4.1 |

Table 5 - Test 1189 - 26.2 Gallon Tank, Drain of 805 g in 30 min - Two-Hour Test - Summary

| Time | Ambient <br> Pressure | Tank <br> Pressure | Tank <br> Temperature | Ambient <br> Temperature | Cumulative <br> Leak |
| :---: | :---: | :---: | :---: | :---: | :---: |
| min | Psi (raw) | psi | ${ }^{\circ}$ F | $g$ |  |
| 0 | 14.1 | 296.0 | 75.9 | 75.7 | 0.0 |
| 10 | 14.1 | 294.4 | 75.6 | 76.2 | 266 |
| 20 | 14.1 | 293.0 | 75.6 | 76.4 | 548 |
| 30 | 14.1 | 291.7 | 75.5 | 75.7 | 782 |
| 40 | 14.1 | 291.8 | 75.6 | 76.8 | 804.3 |
| 50 | 14.1 | 291.8 | 75.7 | 76.3 | 804.3 |
| 60 | 14.1 | 291.8 | 75.6 | 76.6 | 804.3 |
| 70 | 14.1 | 291.8 | 75.6 | 77.1 | 804.3 |
| 80 | 14.1 | 291.7 | 75.6 | 75.8 | 804.3 |
| 90 | 14.1 | 291.7 | 75.6 | 76.2 | 804.3 |
| 100 | 14.1 | 291.7 | 75.5 | 76.9 | 804.3 |
| 110 | 14.1 | 291.7 | 75.5 | 76.8 | 804.3 |
| 120 | 14.1 | 291.7 | 75.5 | 76.7 | 804.3 |



Figure 12 - Test 1189 - 26.2 Gallon Tank, Drain of 805 g Water in 30 min - Pressure, Temperature = f (time)

### 4.14 - Drain 1610 g of Water in 60 min

Run 1209 was a test with the tank filled with water to 40 percent of its capacity and the pressure in the tank set at $292 \mathrm{psig}(+/-2 \%)$, as corrected to normal temperature. The leak was stopped after 60 minutes, but data was collected for 120 minutes to assess pressure stabilization. This run produced the following results (see detailed final results in Table 6, summary data in Table 7, and pressure and temperature graphed as a function of time in Figure 13).

Table 6 - Run 1209 - 26.2 Gallon Tank - Drain of 1610 g Water in 60 min - Two-Hour Test

| RUN No. |  | Tank Total Volume | Water Volume in Tank | $\mathrm{N}_{2}$ Volume in Tank | Weight Water Drain | Change Water Volume | Tank Pres. | Ambient Pres. | Tank Pres. | Ambient Temp | Tank Temp (gas) | Pressure $68^{\circ} \mathrm{F}$ 14.7 psia ambient | Actual $\Delta$ Tank Pres. @NT |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| min | min | L | L | L | g | L | psig | psia | psia | ${ }^{\circ} \mathrm{F}$ | ${ }^{\circ} \mathrm{F}$ | psia | psia |
| $\begin{aligned} & \text { RUN } \\ & 1209 \end{aligned}$ | 0 | 99.2 | 40.1 | 59.1 | 0.0 | 0.000 | 298.6 | 14.1 | 312.7 | 74.0 | 72.0 | 310.3 |  |
| 60 min DRAIN@ 30 min | 30 | 99.2 | 39.3 | 59.9 | 804.5 | -0.805 | 294.3 | 14.0 | 308.3 | 75.0 | 72.0 | 306.0 | -4.3 |
| 60 min DRAIN@ 60 min | 60 | 99.2 | 38.5 | 60.7 | 1608.6 | -1.609 | 290.2 | 14.0 | 304.3 | 75.0 | 72.0 | 302.0 | -8.3 |
| 60 min DRAIN@ 120 min | 120 | 99.2 | 38.5 | 60.7 | 1608.6 | -1.609 | 290.5 | 14.0 | 304.5 | 75.0 | 72.3 | 302.1 | -8.2 |

Table 7 - Run 1209-26.2 Gallon Tank - Drain of 1610 g Water in 60 min - Summary

| Time | Ambient <br> Pressure | Tank <br> Pressure | Tank <br> Temperature | Ambient <br> Temperature | Cumulative <br> Water Drain |
| :---: | :---: | :---: | :---: | :---: | :---: |
| min | psia (raw) | psig | ${ }^{\circ}$ F | ${ }^{\circ} \mathrm{F}$ | g |
| 0 | 14.1 | 298.6 | 72.0 | 73.7 | 0 |
| 10 | 14.1 | 297.0 | 71.9 | 74.1 | 268 |
| 20 | 14.1 | 295.6 | 72.0 | 73.9 | 536 |
| 30 | 14.0 | 294.3 | 72.0 | 73.3 | 755 |
| 40 | 14.0 | 292.9 | 71.9 | 73.3 | 1074 |
| 50 | 14.0 | 291.5 | 71.9 | 73.9 | 1344 |
| 60 | 14.0 | 290.2 | 72.0 | 74.3 | 1606 |
| 70 | 14.0 | 290.4 | 72.1 | 73.8 | 1608 |
| 80 | 14.0 | 290.4 | 72.2 | 74.4 | 1608 |
| 90 | 14.0 | 290.5 | 72.2 | 74.4 | 1608 |
| 100 | 14.0 | 290.5 | 72.3 | 74.5 | 1608 |
| 110 | 14.0 | 290.5 | 72.3 | 74.2 | 1608 |
| 120 | 14.0 | 290.5 | 72.3 | 74.5 | 1608 |



Figure 13 - Test 1209 - Drain of 1610 g of $\mathrm{H}_{2} \mathrm{O}$ in 60 min - Pressure, Temperature $=f$ (time)

### 4.15 - Gaseous Leak Testing

The gaseous leak test was conducted to evaluate the energy loss from gaseous LPG compared to liquid LPG escaping through the same size opening. For the gaseous leak test, we set the needle valve regulating liquid flow to the exact orifice opening that would allow a flow rate of 805 g of water in 30 minutes. This valve and a shutoff valve were then transferred to a tank opening that accessed the gaseous portion of the tank (see Figures 9 and 10).

The tank was filled with 40 percent water and pressurized with nitrogen to $297 \mathrm{psig}+/-2$ percent, as corrected to normal temperature. After pressure stabilization, the shutoff valve was opened for 30 minutes. Initial tank pressure was 297.4 psig and the final tank pressure was 268.3 psig. This represents a loss of 5.2 mol of nitrogen. Assuming a leak of approximately an equivalent number of moles of propane ( $\mathrm{C}_{3} \mathrm{H}_{8}$ ), this gaseous propane leak would then have 28 percent of the "heat content" of 805 g of liquid propane. ${ }^{34}$

## 4.2 - Large Tank Leak Testing Results

### 4.21 - Theoretical Calculations

As mentioned, the test setup was identical to that of the small tank testing, except instead of a 26.2 gallon tank, we evaluated a 41.4 gallon ( 156.7 L ) tank. This tank had the same diameter ( 16 inch OD) as the 26.2 gallon tank, except its overall length was 52 inches instead of 34 inches.

The theoretical calculation results for the 41.4 gallon tank are presented in Table 8. These calculations are for a leak of 805 g of water, at normal temperature and pressure $\left(68.0^{\circ} \mathrm{F}\right.$ and 14.70 psia ambient air pressure), assuming no temperature change in the tank or ambient barometric pressure change during the event.

Table 8 - Theoretical Calculations for Drain of 805 g Water in 30 min - $\mathbf{4 1 . 4}$ Gallon Tank

| RUN No. |  | Tank <br> Total <br> Volume | Water Volume in Tank | $\mathrm{N}_{2}$ <br> Volume in Tank | Weight Water Drain | Change Water Volume | Tank Pres. | Ambient Pres. | Tank Pres. | Ambient Temp | Tank Temp (gas) | Pressure $68{ }^{\circ} \mathrm{F}$ 14.7 psia ambient | Actual $\Delta$ Tank Pres. @NT |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $a / n$ | min | L | L | L | g | L | psig | psia | psia | ${ }^{\circ} \mathrm{F}$ | ${ }^{\circ} \mathrm{F}$ | psia | psia |
| TEST CALC | NA | 156.72 | 63.4 | 93.32 | 0.0 | 0.000 | 292.00 | 14.70 | 306.70 | 68.0 | 68.0 | 306.70 |  |
| $\begin{gathered} \text { ALL } \\ \text { NT } \end{gathered}$ | NA | 156.72 | 62.59 | 94.12 | 804.5 | -0.805 | 289.38 | 14.70 | 304.07 | 68.0 | 68.0 | 304.07 | -2.62 |

[^12]
### 4.22 - Drain of 805 g Water in 30 min

Run 1280 was a test with the tank filled with water to 40 percent of its capacity and the pressure in the tank set at $292 \mathrm{psig}(+/-2 \%)$ as corrected to normal temperature. ${ }^{35}$ The leak was stopped after 30 minutes, but data was collected for 120 minutes to assess pressure stabilization. This run produced the following results (see detailed final results in Table 9, summary data in Table 10, and pressure and temperature graphed as a function of time in Figure 14).

Table 9 - Run 1280-41.4 Gallon Tank - Drain of 805 g Water in 30 min - Two-Hour Test

| RUN No. |  | Tank Total Volume | Water Volume in Tank | $\mathrm{N}_{2}$ Volume in Tank | Weight Water Drain | Change Water Volume | Tank Pres. | Ambient Pres. | Tank Pres. | Ambient Temp | Tank <br> Temp <br> (gas) | Pressure $68^{\circ} \mathrm{F}$ 14.7 psia ambient | Actual $\Delta$ Tank Pres. @NT |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| a/n | min | L | L | L | g | L | psig | psia | psia | ${ }^{\circ} \mathrm{F}$ | ${ }^{\circ} \mathrm{F}$ | psia | psia |
| RUN 1280 | 0 | 156.7 | 63.4 | 93.3 | 0.0 | 0.000 | 295.5 | 14.2 | 309.7 | 74.0 | 71.7 | 307.6 |  |
| 30 min DRAIN@ 30 min | 30 | 156.7 | 62.6 | 94.1 | 805.2 | -0.805 | 292.7 | 14.2 | 306.9 | 75.0 | 71.4 | 304.9 | -2.6 |
| 30 min DRAIN@ 60 min | 60 | 156.7 | 62.6 | 94.1 | 805.2 | -0.805 | 292.9 | 14.2 | 307.1 | 75.0 | 71.8 | 304.9 | -2.6 |
| 30 min DRAIN@ 120 min | 120 | 156.7 | 62.6 | 94.1 | 805.2 | -0.805 | 293.1 | 14.2 | 307.3 | 75.0 | 72.2 | 304.9 | -2.7 |

[^13]Table 10 - Run 1280-41.4 Gallon Tank - Drain of 805 g Water in 30 min - Summary

| Time | Ambient <br> Pressure | Tank <br> Pressure | Tank <br> Temperature | Ambient <br> Temperature | Cumulative <br> Water Drain |
| :---: | :---: | :---: | :---: | :---: | :---: |
| min | psia (raw) | psig | ${ }^{\circ} \mathrm{F}$ | g |  |
| 0 | 14.2 | 295.5 | 71.7 | 71.1 | 0 |
| 10 | 14.2 | 294.5 | 71.5 | 71.5 | 246 |
| 20 | 14.2 | 293.7 | 71.5 | 71.3 | 501 |
| 30 | 14.2 | 292.7 | 71.4 | 71.4 | 805 |
| 30 | 14.2 | 292.7 | 71.4 | 71.4 | 805 |
| 40 | 14.2 | 292.8 | 71.6 | 71.7 | 805 |
| 50 | 14.2 | 292.9 | 71.7 | 71.7 | 805 |
| 60 | 14.2 | 292.9 | 71.8 | 71.9 | 805 |
| 70 | 14.2 | 292.9 | 71.8 | 72.0 | 805 |
| 80 | 14.2 | 293.0 | 71.9 | 71.9 | 805 |
| 90 | 14.2 | 293.0 | 72.0 | 72.1 | 805 |
| 100 | 14.2 | 293.0 | 72.0 | 72.2 | 805 |
| 110 | 14.2 | 293.1 | 72.1 | 72.3 | 805 |
| 120 | 14.2 | 293.1 | 72.2 | 72.5 | 805 |



Figure 14 - Test 1280-41.4 Gallon Tank - 30 min Drain of 805 g of Water - Tank Pressure, Temperature = f (time)

### 4.23 - Drain of 1610 g of Water in $\mathbf{6 0}$ min

Run 1287 was a typical test with the tank filled with water to 40 percent of its capacity and the pressure in the tank at $292 \mathrm{psig}(+/-2 \%)$, as corrected to normal temperature. The leak was stopped after 60 minutes, but data was collected for 120 minutes to assess pressure stabilization. This run produced the following results (see detailed final results in Table 11, summary data in Table 12, and pressure and temperature as a function of time in Figure 15).

Table 11 - Run 1287 - 41.4 Gallon Tank - Drain of 1610 g Water in 60 min

| RUN No. |  | Tank Total Volume | Water Volume in Tank | $\mathrm{N}_{2}$ Volume in Tank | Weight Water Drain | Change Water Volume | Tank Pres. | Ambient Pres. | Tank Pres. | Ambient Temp | $\begin{aligned} & \text { Tank } \\ & \text { Temp } \\ & \text { (gas) } \end{aligned}$ | $\begin{gathered} \text { Pressure } \\ 68^{\circ} \mathrm{F} \\ 14.7 \text { psia } \\ \text { ambient } \end{gathered}$ | Actual <br> $\Delta$ Tank <br> Pres. @NT |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\mathrm{a} / \mathrm{n}$ | min | L | L | L | g | L | psig | psia | psia | ${ }^{\circ} \mathrm{F}$ | ${ }^{\circ} \mathrm{F}$ | psia | psia |
| $\begin{aligned} & \text { RUN } \\ & 1287 \end{aligned}$ | 0 | 156.7 | 63.4 | 93.3 | 0.0 | 0.000 | 293.7 | 14.2 | 307.9 | 74.0 | 71.0 | 306.1 |  |
| 60 min DRAIN@ 30 min | 30 | 156.7 | 62.6 | 94.1 | 804.5 | -0.805 | 291.2 | 14.2 | 305.4 | 75.0 | 71.3 | 303.5 | -2.6 |
| 60 min DRAIN@ 60 min | 60 | 156.7 | 62.6 | 94.1 | 1609.5 | -1.610 | 288.7 | 14.2 | 302.9 | 75.0 | 71.4 | 300.9 | -5.2 |
| 60 min DRAIN@ 120 min | 120 | 156.7 | 62.6 | 94.1 | 1609.5 | -1.610 | 288.8 | 14.2 | 303.0 | 75.0 | 71.4 | 301.0 | -5.1 |

Table 12 - Run 1287 - 41.4 Gallon Tank - Drain of 1610 g Water in 60 min - Summary Data

| Time | Ambient <br> Pressure | Tank <br> Pressure | Tank <br> Temperature | Ambient <br> Temperature | Cumulative <br> Water Drain |
| :---: | :---: | :---: | :---: | :---: | :---: |
| min | psia (raw) | psig | ${ }^{\circ} \mathrm{F}$ | ${ }^{\circ} \mathrm{F}$ | g |
| 0 | 14.2 | 293.7 | 71.0 | 70.6 | 0 |
| 10 | 14.2 | 292.8 | 71.1 | 71.2 | 286 |
| 20 | 14.2 | 292.0 | 71.2 | 71.5 | 517 |
| 30 | 14.2 | 291.2 | 71.3 | 71.6 | 790 |
| 40 | 14.2 | 290.4 | 71.4 | 72.0 | 1055 |
| 50 | 14.2 | 289.5 | 71.4 | 71.8 | 1317 |
| 60 | 14.2 | 288.7 | 71.4 | 72.0 | 1579 |
| 70 | 14.2 | 288.8 | 71.6 | 71.7 | 1609 |
| 80 | 14.2 | 288.8 | 71.7 | 71.7 | 1609 |
| 90 | 14.2 | 288.8 | 71.7 | 71.9 | 1609 |
| 100 | 14.2 | 288.8 | 71.8 | 71.9 | 1609 |
| 110 | 14.2 | 288.8 | 71.8 | 72.1 | 1609 |
| 120 | 14.2 | 288.8 | 71.9 | 71.9 | 1609 |



Figure 15 - Test 1287 - 41.4 Gallon Tank - Drain of 1610 g of $\mathrm{H}_{2} \mathrm{O}$ in 60 min , Tank Pressure, Temp. = f (time)
Data from other test runs to assess the repeatability and consistency of the tests are presented in Appendix E.

## 5.0 - FINDINGS AND DISCUSSION

## 5.1 - Findings

- Finding 1: For smaller tanks, measuring pressure drop to calculate fluid leakage is within the accuracy of the measurement system. As tanks get larger, the pressure drop caused by the loss of 805 g of fluid diminishes. Thus recording the pressure drop over a longer period of time, such as 60 minutes, would measure any leak rate more accurately.
- Finding 2: The temperature of the gaseous portion of the tank should be recorded so that the measured tank pressure can be corrected for temperature change. The Canadian standard specifies temperature corrected pressure.
- Finding 3: Gaseous LPG that leaked over a 30-minute period has 28 percent of the "energy content of liquid LPG leaked" through the same size hole (damage) for the same period of time. The Canadian specification of fluid loss and/or reduction in tank pressure does not include rolling the vehicle.
- Finding 4: During the laboratory bench testing, the leak was "shut off" then the tank pressure and tank temperature were allowed to stabilize. This cannot be done during a crash test since a leak will continue as long as the system is pressurized. Detailed time stamps are necessary to evaluate the finishing pressure after a " 30 -minute leak" or a " 60 -minute leak" since the system will continue to lose pressure. Then, based on temperature change, actual pressure loss at NTP, etc. must be estimated. ${ }^{36}$
- Finding 5: Absolute pressure can be measured instead of gauge pressure to eliminate the effects of atmospheric pressure changes during testing.
- Finding 5A: A high stability, high reliability, broad temperature compensation range pressure transducer with NIST traceable calibration helps to minimize potential drift in output values. ${ }^{37}$
- Finding 6: Pressure transducers can be insulated to reduce the effects of temperature changes on the reading.

[^14]
## 5.2 - Discussion

The primary challenge for evaluating the fuel system integrity of LPG-fueled vehicles is that LPG exists as both a liquid and a gas in the tank, which is not practical to exactly duplicate during a crash test. However, use of pressurized water and nitrogen was shown to be a reasonable simulation. Loss of 'fuel' can be determined by post-crash test measurements of both pressure drop in the tank and volume of liquid lost. Depending on the number of leaks and their locations, a measured post-test pressure drop may be due to a gaseous leak, a liquid leak (pressure drop due to increased gas volume in tank from loss of liquid), or both.

By measuring both pressure drop and volume of liquid lost, the portion of the pressure drop (if any) due solely to a liquid loss can be calculated and thereby separated from the portion of pressure drop (if any) due to a gaseous leak. Since the energy content of liquid and gaseous propane escaping though the same size hole over the same period of time are not equal, equivalent energy should be calculated separately to determine the energy content of a gaseous leak and a liquid leak. The combination of these is the total energy content lost.

The pressure drop due to only a marginal loss of liquid ( 805 g of water) is relatively small compared to the total tank pressure, particularly for larger LPG tanks. This study showed that pressure transducers and data acquisition systems are available that can accurately measure these pressure drops, but care must be taken to account for ambient pressure and temperature changes.

## 6.0 - ADDITIONAL TESTS

Specialized test results can be found in Appendix B including Appendix B1, variable leakage rate tests, Appendix B2, a rapid pressure drop leak test, Appendix B3 - evaluation of tank fill and stabilization procedures, Appendix B4, tank temperature and ambient temperature relationships, Appendix B5, large pressure drop volume evaluations, and Appendix B6, large tank overrun tests.

## Appendix A - General Information

Table 13 - Acronym List

| Acronym |  |
| :--- | :--- |
| Btu | British thermal unit |
| CFR | Code of Federal Regulations |
| DOT | Department of Transportation |
| DUT | device under test |
| FMVSS | Federal Motor Vehicle Safety Standard |
| FOM | figure of merit |
| FS | full scale |
| GM | General Motors Company |
| HHV | high heating value |
| ISSN | International Standard Serial Number |
| IUPAC | International Union of Pure and Applied Chemistry |
| LHV | liquefied petroleum gas |
| LPG | Maximum working pressure |
| MAWP | not apply/not applicable |
| na | National Highway Traffic Safety Administration |
| NHTSA | National Institute of Standards and Technology |
| NIST | not recorded |
| nr | normal temperature and pressure |
| NTP | NHTSA Vehicle Safety |
| NVS | original equipment |
| OE | original equipment manufacturer |
| OEM | pounds per square inch absolute |
| psia | pounds per square inch gauge (gauge pressure) |
| psig | root of the mean squared |
| RMS | root of the sum squared |
| RSS | standard temperature and pressure |
| STP | to be determined |
| TBD | Transport Canada |
| TC | test plan |
| TP | Transportation Research Center |
| TRC | Vehicle Research and Test Center |
| VRTC |  |
|  |  |

## Appendix A2 - References Consulted

Table 14 - References Consulted

| Number | Report Name |
| :---: | :---: |
| 0894-1777/88 | Describing the Uncertainties in Experimental Results, Robert J. Moffat, Professor of Mechanical Engineering, Stanford University, Stanford, California, Experimental Thermal and Fluid Science 1988; 1:3-17, 1988 by Elsevier Science Publishing Co., Inc. New York, NY 10017 |
| 49 CFR Part 571, Sec. 301 | Subpart B-Federal Motor Vehicle Safety Standards: Standard No. 301; Fuel system integrity. |
| 49 CFR Part 571, Sec. 303 | Subpart B-Federal Motor Vehicle Safety Standards: Standard No. 303; Fuel system integrity of compressed natural gas vehicles (571.303) |
| CEC-600-2007-002-D | FULL FUEL CYCLE ASSESSMENT WELL TO Tank ENERGY INPUTS, EMISSIONS, AND WATER IMPACTS, Prepared For: California Energy Commission, February 2007 |
| ISSN 1368-655 | A Beginner's Guide to Uncertainty of Measurement, Measurement Good Practice Guide No. 11 (Issue 2), Stephanie Bell, Centre for Basic, Thermal and Length Metrology, National Physical Laboratory, August 1999, Issue 2 with amendments March 2001 |
| ISSN 1368-6550 | Estimating Uncertainties in Testing, An Intermediate Guide to Estimating and Reporting Uncertainty of Measurement in Testing, Measurement Good Practice Guide No. 36, Keith Birch, British Measurement and Testing Association, Crown Copyright 2001 |
| GM Petition | GM Petition FMVSS No. 303 Upgrade Briefing, NHTSA, MRT Briefing, September 18, 2012 |
| NIST Technical Note 1297 | Guidelines for Evaluating and Expressing the Uncertainty of NIST Measurement Results, Barry N. Taylor and Chris E. Kuyatt, United States Department of Commerce Technology Administration, National Institute of Standards and Technology, September 1994 |
| NIST Standard Reference Database Number 69 | E. W. Lemmon, M. O. McLinden and D. G. Friend, "Thermophysical Properties of Fluid Systems" in NIST Chemistry WebBook, NIST Standard Reference Database Number 69, Eds. P. J. Linstrom and W. G. Mallard, National Institute of Standards and Technology, Gaithersburg MD, 20899, http://webbook.nist.gov |
| TC 301.1 | Reference Transport Canada, Standards and Regulations Division, Test Method 301.1, LPG Fuel System Integrity, Revised: February 8, 2001, with update to SOR/2008, 104 April 16, 2008, 301.1 Section (2) (a) and (b). |
| TP-301-04, (Jan 17, 2007) | U. S. DEPARTMENT OF TRANSPORTATION, NATIONAL HIGHWAY TRAFFIC SAFETY ADMINISTRATION, LABORATORY TEST PROCEDURE FOR FMVSS 212 Windshield Mounting FMVSS 219 Windshield Zone Intrusion FMVSS 301 Fuel System Integrity |
| TP-303.00 (June 30, 1995) | U. S. DEPARTMENT OF TRANSPORTATION, NATIONAL HIGHWAY TRAFFIC SAFETY ADMINISTRATION, LABORATORY TEST PROCEDURE FOR FMVSS 212 Windshield Mounting, FMVSS 219 Windshield Zone Intrusion, FMVSS 303 Fuel System Integrity of Compressed Natural Gas (CNG) Vehicles |
| VRTC-83-0307 (July 1994) | VRTC Event Report: Compressed Natural Gas Fuel System Integrity |

## Appendix A3 - Equipment and Instrumentation

Table 15 - Equipment and Instrumentation

| No. | Description | Make | Model | Serial No. | Size | Application |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | LPG Tank | Manchester | P/N 2380417 | 1737315 | 26.2 gal | Test Article |
| 1 a | LPG Tank | Manchester | P/N 20067 | 1735147 | 41.4 gal | Test Article |
| 2 | LPG Fill Water | Tank Full | na | na | 88.45 lbm | LPG equivalent |
| 3 | Pressure <br> Transducer | Omega | PX409-500G5V | 442527 | $0-500 \mathrm{psig}$ | Tank pressure |
| 4 | Pressure <br> Transducer | Omega | PX302-300GV | 140210 | $0-300 \mathrm{psig}$ | Tank pressure |
| 4 a | Pressure <br> Transducer | Omega | PX302-015AV | 121213 | $0-15 \mathrm{psia}$ | Ambient Air <br> Pressure |
| 5 | Thermocouple | Applied Sensor <br> Technologies | Analog Devices <br> $3 B 47$ J-type | na | na | Tank <br> Temperature |
| 6 | Needle Valve | na | na | $\sim 0-500$ <br> $\mathrm{~g} / \mathrm{min}$ | Flow Control |  |
| 7 | Thermocouple | J-type thermocouple <br> wire | Analog Devices <br> 3B47 J-type | na | na | Ambient <br> Temperature |
| 8 | Digital Scale | AND | EJ-6100 | $5 A 2833373$ | 6100 g | Water mass <br> flow rate |
| 9 | Mass Flow <br> Meter | Omega | FMA1827A | $360936-3$ | $0-40 \mathrm{~L} / \mathrm{min}$ | Gas flow rate |
| 10 | Pressure <br> Calibrator | Omega | PCL341 | 9306073 | $0-1000 \mathrm{psi}$ | Tank Pressure |

## Appendix B - Special Tests

## Appendix B1 - Variable Leakage Rate Tests

For selected test runs, the leakage rate was varied from $23.5 \mathrm{~g} / \mathrm{min}$ to $456.5 \mathrm{~g} / \mathrm{min}$. Changes in pressure and temperature were evaluated as a function of leakage rate. After the leaks were stopped, the pressure and temperature recovered from their initial drops to "stable" values were also evaluated ${ }^{38}$ See Table 16 for a tabulation of leakage and recovery values and Figures 16 and 17 for plotting of the results.

Table 16-26.6 Gallon Tank - Pressure and Temperature Change $=\mathbf{f}$ (leakage rate)

| Test <br> Run | Slow/Regular/Fast | Rate | Max <br> Cressure <br> Change | Max <br> Temp <br> Change | Pressure <br> Recover | Temp <br> Recover |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\mathrm{a} / \mathrm{n}$ | a/n | $\mathrm{g} / \mathrm{min}$ | psig | ${ }^{\circ} \mathrm{F}$ | psig | ${ }^{\circ} \mathrm{F}$ |
| 1210 | Fast | 51.27 | -4.39 | -0.23 | 0.21 | 0.15 |
| 1173 | Fast | 456.54 | -5.32 | -0.75 | 0.81 | 0.27 |
| 1169 | Slow | 23.52 | -4.19 | -0.09 | 0.13 | 0.19 |
| 1172 | Regular | 26.62 | -4.27 | -0.21 | 0.13 | 0.05 |
| 1201 | Regular | 26.64 | -5.12 | -0.14 | 0.17 | 0.20 |




Figure 16 - 26.2 Gallon Tank - Initial Pressure and Temperature Drop = $\mathbf{f}$ (leakage rate) ${ }^{39}$

[^15]

Figure 17-26.2 Gallon Tank - Recovery of Initial Pressure and Temperature Drop = f(leakage rate)

## Appendix B2 - Rapid Pressure Drop Leak Test

The needle valve on the fluid drain was opened to the maximum extent possible. While the resultant gauge pressure was affected to some extent, the temperature-corrected absolute pressure after stabilization was close to the theoretical value (see Table 17 for input data, Table 18 for detailed final results, and Figure 18 for a graph of pressure and temperature as a function of time).

Table 17 - Run 1173-26.2 Gallon Tank - Drain of 805 g Water in 1.78 min - Input Data

| Run |  | Action | Tank <br> Total <br> Volume | Water <br> Volume <br> in Tank | Nitrogen Volume in Tank | Weight Water Drain | Change Water Volume | Change Nitrogen Volume in Tank | Tank Pres. |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| min | min | $a / n$ | L | L | L | g | L | L | psig |
| RUN1173 | 0 | START: STABILIZE; TEMPERATURE CORRECTED +/- $2 \%$ | 99.2 | 40.1 | 59.1 | 0.0 | 0.000 | 0.000 | 296.3 |
| $\begin{aligned} & \text { FAST } \\ & \text { DRAIN } \end{aligned}$ | 1.78 | FAST Drain Water; Drained 816.6 g Water in 1:47.02 ( 1.78 min ) | 99.2 | 39.3 | 59.9 | 816.6 | -0.817 | 0.817 | 291.5 |

Table 18 - Run 1173-26.2 Gallon Tank - Drain of 805 g Water in 1.78 min - Detailed Results

| Run |  | Ambient Temp | Ambient Pres. | Tank Temp (gas) | Actual Pressure $@ 68{ }^{\circ} \mathrm{F}$ NT | Actual $\Delta$ Tank Pres. @NT | $\Delta$ Tank Raw Gauge Pres. |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| min | min | ${ }^{\circ} \mathrm{F}$ | psia | ${ }^{\circ} \mathrm{F}$ | psia | psia | psig |
| RUN1173 | 0 | 74.0 | 14.3 | 76.6 | 305.6 |  |  |
| FAST DRAIN | 1.78 | 75.0 | 14.3 | 75.6 | 301.5 | -4.2 | -4.8 |



Figure 18 - Test 1173-26.2 Gal Tank - Fast Drain - Tank Pressure and Temperature $=\mathbf{f}$ (time)

## Appendix B3 - Evaluation of Tank Fill Procedures

In order to develop an optimum fill procedure for the tanks, a number of tests were run to evaluate final pretest fill temperature and pressure as a function of initial fill pressure prior to cool down. Based in part on the methodologies of the CNG tank fill procedure, we found that to achieve a desired starting pressure of $292 \mathrm{psig},{ }^{40}$ filling the tank to 310 psig , allowing it to stabilize for one hour, refilling it to 300 psig , and then allowing it to stabilize for an additional hour was sufficient to achieve the desired starting pressure. ${ }^{41}$

[^16]
## Appendix B4 - Tank Temperature and Ambient Temperature Relationships

In order to develop an understanding of how changes in ambient temperature might affect changes in tank temperature during an actual FMVSS test set up and execution, data was taken over an extended period of time during which ambient temperature fluctuated due to normal daily heating and cooling. These temperatures are measured in a test bay. As can be seen in Figure 19, changes in tank temperature typically lagged the changes in ambient temperature, and the tank temperature was typically below ambient temperature $\left(0.94^{\circ} \mathrm{F}\right.$ maximum). ${ }^{4}$


Figure 19-26.2 Gallon Tank - Internal Tank Temperature as a Function of Ambient Temperature

[^17]
## Appendix B5 - Large Pressure Drop Volume Evaluations

Additional calculations were conducted to determine what amount of liquid (water) would have to be drained (in 30 min ) to achieve pressure drop of 15.5 psi over 60 minutes. We used the normal temperature and pressure leak rate calculations to evaluate the approximate amount of change in water volume that would cause a 15.5 psia pressure drop. See Table 19 and Table 20 for details on the 26.2 gallon and the 41.4 gallon tank, respectively.

Table 19 - Calculated Drain of Water Small Tank for 15.5 psi Pressure Drop@Exact NT ${ }^{43}$

| RUN No. |  | Tank Total Volume | Water Volume in Tank | Nitrogen Volume in Tank | Weight Water Drain | Change Water Volume | Tank Pres. | Ambient Pres. | Tank Pres. | Ambient Temp | Tank <br> Temp <br> (gas) | New Pressure @68 ${ }^{\circ} \mathrm{F}$ NT | Actual <br> $\Delta$ Tank Pres. @NT |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| min | min | L | L | L | g | L | psig | psia | psia | ${ }^{\circ} \mathrm{F}$ | ${ }^{\circ} \mathrm{F}$ | psia | psia |
| TEST CALC | NA | 99.2 | 40.1 | 59.1 | 0.0 | 0.000 | 292.0 | 14.7 | 306.7 | 68.0 | 68.0 | 306.7 |  |
| $\begin{aligned} & \text { ALL } \\ & \text { NT } \end{aligned}$ | NA | 99.2 | 37.0 | 62.2 | 3145.0 | -3.145 | 276.5 | 14.7 | 291.2 | 68.0 | 68.0 | 291.2 | -15.5 |

Table 20 - Calculated Drain of Water Large Tank for 15.5 psi Pressure Drop@Exact NT

| RUN No. |  | Tank Total Volume | Water Volume in Tank | Nitrogen Volume in Tank | Weight Water Drain | Change Water Volume | Tank Pres. | Ambient Pres. | Tank Pres. | Ambient Temp | Tank Temp (gas) | New Pressure @68 ${ }^{\circ} \mathrm{F}$ NT | Actual $\Delta$ Tank Pres. @NT |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| a/n | min | L | L | L | g | L | psig | psia | psia | ${ }^{\circ} \mathrm{F}$ | ${ }^{\circ} \mathrm{F}$ | psia | psia |
| TEST CALC | NA | 156.7 | 63.4 | 93.3 | 0.0 | 0.000 | 292.0 | 14.7 | 306.7 | 68.0 | 68.0 | 306.7 |  |
| ALL NT | NA | 156.7 | 58.4 | 98.3 | 4967.0 | -4.967 | 276.5 | 14.7 | 291.2 | 68.0 | 68.0 | 291.2 | -15.5 |

Assuming standard temperature and absolute pressure, ${ }^{44}$ a drain of $3.145 \mathrm{~L}(0.83 \mathrm{gal})$ from the 26.2 gallon tank would produce a pressure drop of 15.5 psig. Similarly for the 41.4 gallon tank, a drain of 4.967 L $(1.31 \mathrm{gal})$ would produce a pressure drop of $15.5 \mathrm{psi} .{ }^{45}$

[^18]
## Appendix B6 - Large Tank Overrun Test

Since in an actual LPG vehicle system test, the leak will not stop at a specific time, we must determine how stable measurements are at a specific time if the tank or fuel system breach continues to occur beyond that time. For this test, we used a leak rate of approximately 26.8 $\mathrm{g} / \mathrm{min}$ and took measurements at 60 minutes, but we allowed the leak to continue after 60 minutes. We picked 60 minutes because at least for larger tanks, a 30 -minute leak of 2.6 psia may be difficult to measure (see Table 21 for detailed final results, Table 22 for summary data, and Figure 20 for a graph of pressure and temperature as a function of time).

Table 21 - 41.4 Gallon Tank - Drain of 1900 g at $26.8 \mathrm{~g} / \mathrm{min}$ for 70 min - Stop at 120 min

| RUN No. |  | Tank Total Volume | Water Volume in Tank | Nitrogen Volume in Tank | Weight Water Drain | Change Water Volume | Tank Pres. | Ambient Pres. | Tank Pres. | Ambient Temp | Tank Temp (gas) | $\begin{gathered} \text { Pressure } \\ 68^{\circ} \mathrm{F} \\ 14.7 \\ \text { psia } \\ \text { ambient } \end{gathered}$ | Actual ATank Pres. @NT |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\mathrm{a} / \mathrm{n}$ | min | L | L | L | g | L | psig | psia | psia | ${ }^{\circ} \mathrm{F}$ | ${ }^{\circ} \mathrm{F}$ | psia | psia |
| $\begin{aligned} & \text { RUN } \\ & 1303 \end{aligned}$ | 0 | 156.7 | 63.4 | 93.3 | 0.0 | 0.000 | 293.9 | 14.2 | 308.1 | 74.0 | 78.6 | 302.0 |  |
| 70 min Drain @ 30 min | 30 | 156.7 | 62.6 | 94.1 | 804.5 | -0.805 | 291.3 | 14.2 | 305.5 | 75.0 | 78.7 | 299.5 | -2.6 |
| 70 min Drain @ 60 min | 60 | 156.7 | 61.8 | 94.9 | 1608.6 | -1.609 | 288.8 | 14.2 | 303.0 | 75.0 | 79.0 | 296.8 | -5.2 |
| 70 min Drain @ 70 min | 60 | 156.7 | 61.5 | 95.2 | 1900.8 | -1.901 | 288.0 | 14.2 | 302.2 | 75.0 | 79.2 | 295.9 | -6.1 |
| 70 min Drain @ 120 min | 60 | 156.7 | 61.5 | 95.2 | 1900.8 | -1.901 | 288.5 | 14.2 | 302.7 | 75.0 | 80.0 | 295.9 | -6.1 |

Table 22 - Run 1303-41.4 Gallon Tank - Drain of 1901 g Water in 70 min - Summary

| Time | Ambient <br> Pressure | Tank <br> Pressure | Tank <br> Temperature | Ambient <br> Temperature | Cumulative <br> Water Drain |
| :---: | :---: | :---: | :---: | :---: | :---: |
| min | psia (raw) | psig | ${ }^{\circ} \mathrm{F}$ | ${ }^{\circ} \mathrm{F}$ | g |
| 0 | 14.2 | 293.9 | 78.6 | 82.2 | 0 |
| 10 | 14.2 | 293.0 | 78.6 | 82.5 | 264 |
| 20 | 14.2 | 292.1 | 78.6 | 82.9 | 518 |
| 30 | 14.2 | 291.3 | 78.7 | 83.2 | 794 |
| 40 | 14.2 | 290.5 | 78.8 | 83.9 | 1088 |
| 50 | 14.2 | 289.6 | 78.9 | 84.1 | 1347 |
| 60 | 14.2 | 288.8 | 79.0 | 84.1 | 1608 |
| 70 | 14.2 | 288.1 | 79.2 | 84.1 | 1901 |
| 80 | 14.2 | 288.0 | 79.2 | 84.0 | 1901 |
| 90 | 14.2 | 288.2 | 79.5 | 84.5 | 1901 |
| 100 | 14.2 | 288.4 | 79.8 | 84.3 | 1901 |
| 110 | 14.2 | 288.4 | 79.9 | 85.0 | 1901 |
| 120 | 14.2 | 288.5 | 80.0 | 84.3 | 1901 |



Tank Temperature ( ${ }^{\circ}$ F) RED LINE

Figure 20 - 41.4 Gallon Tank - Drain of 1900 g Water in 70 min

## Appendix C - Calculations Details

## Appendix C1 - Calculations LPG Equivalent and Energy Content Values

Gasoline gallon equivalent of LPG to duplicate total energy fuel leakage is as follows:

| Fuel Leakage |  |  |  |
| :--- | :--- | ---: | :--- |
| Gasoline | $=$ | 28 | g |
| Gasoline | $=$ | 142 | g |
| Gasoline | $=$ | 700 | g |
| Gasoline | $=$ | 870 | g |

§571.301 Standard No. 301; Fuel system integrity states:
Initial fuel loss
5 min fuel loss
28 g min fuel loss
Total fuel loss

| Energy Content of Various Fuels |  |  |
| :--- | :--- | ---: |
| Gasoline | $=$ | 20800 |
| Gasoline | $=$ | 19200 |
| Methane | $=$ | 23900 |
| Methane | $=$ | 21500 |
| Propane | $=$ | 21669 |
| Propane | $=$ | 19950 |
| Propane | $=$ | 21548 |
| Propane | $=$ | 21591 |


| Energy Content of Various Fuels |  | Per gram |  |  |
| :--- | :--- | :--- | :--- | :--- |
| Gasoline | $=$ | 45.86 | $\mathrm{Btu} / \mathrm{g}$ | HHV CEC-600-2007-002-D, CA Energy Commission $(6.00 \mathrm{lbm} / \mathrm{gal})$ |
| Gasoline | $=$ | 42.33 | $\mathrm{Btu} / \mathrm{g}$ | LHV CEC-600-2007-002-D, CA Energy Commission $(6.00 \mathrm{lbm} / \mathrm{gal})$ |
| Methane | $=$ | 52.69 | $\mathrm{Btu} / \mathrm{g}$ | HHV CEC-600-2007-002-D, CA Energy Commission $(4.31 \mathrm{lb} / 100 \mathrm{scf})$ |
| Methane | $=$ | 47.39 | $\mathrm{Btu} / \mathrm{g}$ | LHV CEC-600-2007-002-D, CA Energy Commission $(4.31 \mathrm{lb} / 100 \mathrm{scf})$ |
| Propane | $=$ | 47.77 | $\mathrm{Btu} / \mathrm{g}$ | HHV CEC-600-2007-002-D, CA Energy Commission $(4.26 \mathrm{lbm} / \mathrm{gal})$ |
| Propane | $=$ | 43.98 | $\mathrm{Btu} / \mathrm{g}$ | LHV CEC-600-2007-002-D, CA Energy Commission $(4.26 \mathrm{lbm} / \mathrm{gal})$ |
| Propane | $=$ | 47.50 | $\mathrm{Btu} / \mathrm{g}$ | NFPA 58, Liquefied Petroleum Gas Code, 1998 Ed., Table B-1.2 |
| Propane | $=$ | 47.60 | $\mathrm{Btu} / \mathrm{g}$ | http://www.propane101.com/aboutpropane.htm [10/09/2014 12:48] |
| Methane | $=$ | 46.36 | $\mathrm{Btu} / \mathrm{g}$ | from GM Petition Chart |
| Gasoline | $=$ | 44.09 | $\mathrm{Btu} / \mathrm{g}$ | from GM Petition Chart |
| Propane | $=$ | 47.70 | $\mathrm{Btu} / \mathrm{g}$ | from GM Petition Chart |


|  | Energy Leak Rate of LPG liquid | equivalent to Gasoline Leak Rate |  |
| :--- | :--- | ---: | :--- |
| LPG | $=$ | 25.9 | g |

## Appendix C2 - Calculations of Gaseous Leak Rates

Calculation results are as follows. We have:

$$
\begin{aligned}
& P * V=Z * n * R * T \\
& \left(P_{t(N 2 i)} * V_{t(N 2 i)} /\left(Z_{(N 2)} * R * T_{(N 2 i))}\right)=n_{(N 2 i)}\right. \text { moles } \\
& \left(P_{t(N 2 f)} * V_{t(N 2 f)} /\left(Z_{(N 2)} * R * T_{(N 2 f)}\right)=n_{(N 2 f)}\right. \text { moles }
\end{aligned}
$$

where:

| $P_{t(N 2 i)}$ | is | $\mathrm{N}_{2}$ pressure tank initial |
| :--- | :--- | :--- |
| $P_{t(N 2 f)}$ | is | $\mathrm{N}_{2}$ pressure tank final |
| $V_{t(N 2 i)}$ | is | $\mathrm{N}_{2}$ volume tank initial |
| $V_{t(N 2 f)}$ | is | $\mathrm{N}_{2}$ volume tank final |
| $T_{\text {(N2i) }}$ | is | $\mathrm{N}_{2}$ temperature initial |
| $T_{(N 2 f)}$ | is | $\mathrm{N}_{2}$ temperature final |
| $N_{(N 2 i)}$ | is | $\mathrm{N}_{2}$ number of moles initial |
| $N_{(N 2 f)}$ | is | $\mathrm{N}_{2}$ number of moles final |
| $Z_{\text {(N2) }}$ | is | $\mathrm{Nitrogen} \mathrm{compressibility} \mathrm{factor}^{R}$ |
| $R$ | is | molar gas constant |


| in | kPa absolute |
| :--- | :--- |
| in kPa absolute |  |
| in liters |  |
| in liters |  |
| in K |  |
| in K |  |
| in mol |  |
| in mol |  |
| in $1 / 1$ or dimensionless ${ }^{46}$ |  |
| in $(\mathrm{L} * \mathrm{kPa}) /\left(\right.$ mol $\left.^{4} \mathrm{~K}\right)$ |  |

Where measured values and units are as follows:

| $P_{t(N 2 i)}$ | is | $\mathrm{N}_{2}$ gauge pressure tank initial | $=$ | 297.4 | psig |
| :---: | :---: | :---: | :---: | :---: | :---: |
| $P_{t(N 2 f)}$ | is | $\mathrm{N}_{2}$ gauge pressure tank final | = | 268.3 | psig |
| $V_{(N 2 i)}$ | is | $\mathrm{N}_{2}$ volume tank initial | $=$ | 59.0 | liter |
| $V_{(N 2 f)}$ | is | $\mathrm{N}_{2}$ volume tank final | $=$ | 59.0 | liter |
| $P_{a(i)}$ | is | atmospheric pressure (raw) initial | $=$ | 14.1 | psia (raw) |
| $P_{a(f)}$ | is | atmospheric pressure (raw) final | $=$ | 14.1 | psia (raw) |
| $P_{t(N 2 i)}$ | is | $\mathrm{N}_{2}$ absolute pressure tank initial | $=$ | 311.5 | psia |
| $P_{t(N 2 f)}$ | is | $\mathrm{N}_{2}$ absolute pressure tank final | $=$ | 282.3 | psia |
| $k P a / p s i$ |  | conversion | $=$ | 6.8948 | kPa/psi |
| $P_{t(N 2 i)}$ | is | $\mathrm{N}_{2}$ absolute pressure tank initial | $=$ | 2147.5 | kPa absolute |
| $P_{t(N 2 f)}$ | is | $\mathrm{N}_{2}$ absolute pressure tank final | $=$ | 1946.7 | kPa absolute |
| $T_{(N 2 i)}$ | is | $\mathrm{N}_{2}$ temperature initial in Fahrenheit | $=$ | 69.8 | F |
| $T_{(N 2 f)}$ | is | $\mathrm{N}_{2}$ temperature final in Fahrenheit | = | 73.7 | F |
| $T_{(N 2 i)}$ | is | $\mathrm{N}_{2}$ temperature initial in degrees Kelvin | $=$ | 294.2 | K |
| $T_{\text {(N2f) }}$ | is | $\mathrm{N}_{2}$ temperature final in degrees Kelvin | = | 296.3 | K |
| $R$ | is | molar gas constant | = | 8.314 | kPa) / $(\mathrm{mol} * \operatorname{degK})$ |

Therefore:

| $n_{(N 2 i)}$ | is | number of $\mathrm{N}_{2}$ moles initial | $=$ | 51.8 |
| :--- | :--- | :--- | :--- | :--- |
| $n_{(N 2 f)}$ | is | number of $\mathrm{N}_{2}$ moles final | $=$ | mol |
| $\Delta n_{(N 2)}$ | is | change moles $\mathrm{N}_{2}$ in tank | $=$ | mol |
|  |  |  | 5.2 | mol |

For an approximate estimate assume same number of moles of $\mathrm{C}_{3} \mathrm{H}_{8}$ as moles $\mathrm{N}_{2}$ passing through orifice: ${ }^{47}$

| $U_{(\text {(C3H8)/g }}$ | is | LPG energy/unit weight | $=$ | 47.7 | BTU/g |
| :---: | :---: | :---: | :---: | :---: | :---: |
| $M W_{\text {(С3Н8) }}$ | is | LPG Molecular Weight |  | 44.1 | $\mathrm{g} / \mathrm{mol}$ |
| $\mathrm{C}_{3} \mathrm{H}_{8}$ (loss) | is | $\mathrm{C}_{3} \mathrm{H}_{8}$ for gas leak |  | 228.3 | g |
| $U_{\text {(С3Н8) }}$ | is | Heat $\mathrm{C}_{3} \mathrm{H}_{8}$ gas leak |  | 10891.4 | BTU |
| $\mathrm{C}_{3} \mathrm{H}_{8}$ (loss) | is | $\mathrm{C}_{3} \mathrm{H}_{8}$ allowable |  | 805 | g |
| Gas vs. Liquid \% |  | "Ratio gas vs liquid leak |  | 28 | percent |

[^19]
## Appendix D - Tank Technical Data

Specifications and other technical data for the LPG tanks used in this study are found in Table 23 and in Figures 21 through 24. ${ }^{48} 49505152$ The Manchester autogas tanks used in these tests were manufactured to ASME Section VIII, Division I Code and meet NFPA \# 58 standards for engine fuel applications. Liquid, vapor, or dual service (liquid and vapor) type tanks are options for autogas tanks. Standard features include 80 percent stop fill device and seal welded stainless steel ASME nameplate. Direct or remote fill options and engineered strap brackets or custom designs are available. All tanks are designed for 312 psi maximum allowable working pressure. The tanks have undergone multiple testing procedures, including being pneumatically tested with dry air to eliminate water and corrosion which might plug water drain ports, etc.

The small 26.2 gallon tank, $\mathrm{P} / \mathrm{N} 2380417$, is used for mounting in car trunks. It weighs 98 lb and fits into Ford Taurus, Chevrolet Impala, Chevrolet Caprice, and Dodge Charger vehicles for most model years (see Figures 21 and 22). The larger 41.4 gallon tank, $\mathrm{P} / \mathrm{N} 20067$, is used as a large capacity under cap or cover mount tank for small pick-up trucks. It weighs 152 lb (see Figures 23 and 24). ${ }^{53}$

Table 23 - Detailed Tank Specifications

|  | $\dot{ভ}$ | 듬 $\stackrel{\square}{\top}$ |  |  | MAWP | MAWP <br> Temp <br> Spec. | $\mathrm{C}_{3} \mathrm{H}_{8}$ <br> Cap. <br> 80\% | Weight $\mathrm{C}_{3} \mathrm{H}_{8}$ | Weight $\mathrm{C}_{3} \mathrm{H}_{8}$ | Water Require | Water Require $20^{\circ} \mathrm{C}$ | Free Vol in Tank |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| P/N | inch | inch | gal | L | psi | ${ }^{\circ} \mathrm{F}$ | gal | lbm | kg | kg | L | L |
| 2380417 | 16 | 34 | 26.2 | 99.18 | 312 | 650 | 20.96 | 88.45 | 40.12 | 40.12 | 40.19 | 58.98 |
| 20067 | 16 | 52 | 41.4 | 156.72 | 312 | 650 | 33.12 | 139.77 | 63.40 | 63.40 | 63.51 | 93.20 |

[^20]
S.G. ADAPT.


| SPECIFICATONS |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
|  | SH. <br> HD. <br> GAL. 26.2 SA. 12.7 <br> WT. 98\# CU.FT. 3.5 <br> CORR ALI SH HD COOE ASME SECT. VIII, DN. 1 <br> PER DATE OF DWG/LAST REV |  |  |  |
|  |  |  | ORF | NK |
|  |  | WN Er | Date | SCAE: NONE |
| WELD DETALLS: M-1932 <br> STD. TOLERANCES (UNLESS | OTHERWISE NOTED): M-2461 | ${ }^{\text {2. }{ }^{\text {PV }} \text { WL }}$ | DAEE | 2380417 |

Figure 21 - 26.2 Gallon Manchester LPG Tank PN 2380417


Figure 22-26.2 Gallon Manchester LPG Tank Serial No. Plate


Figure 23 - 41.4 Gallon Manchester LPG Tank PN 20067


Figure 24-41.4 Gallon Manchester LPG Tank Serial No. Plate

## Appendix E - Other Data Runs

Other test runs to assess repeatability and consistency when correcting results to normal temperature and pressure are as follows:
Table 24 - Summary of Regular Test Data for Small Tank (26.2 gallon) ${ }^{\text {54 }}$

| Test Run/Action | Tank <br> Total <br> Volume | Water Volume in Tank | $\mathrm{N}_{2}$ <br> Volume in Tank | Weight Water Drain | Change Water Volume | Tank Pres. | Ambient Pres. | Tank Pres. | Ambient Temp | Tank Temp (gas) | Pressure $68{ }^{\circ} \mathrm{F}$ 14.7 psia ambient | Actual $\Delta$ Tank Pressure @NT |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\mathrm{a} / \mathrm{n}$ | L | L | L | g | L | psig | psia | psia | ${ }^{\circ} \mathrm{F}$ | ${ }^{\circ} \mathrm{F}$ | psia | psia |
| RUN 1114 DRAIN 30 min TOTAL DRAIN, DATA @60 min | 99.2 | 39.3 | 59.9 | 804.7 | -0.805 | 289.9 | 14.1 | 304.0 | 75.0 | 72.8 | 301.3 | -4.0 |
| RUN 1125 DRAIN 60 min TOTAL DRAIN, DATA @30 min | 99.2 | 40.1 | 59.9 | 806.5 | 0.000 | 292.8 | 14.2 | 307.0 | 75.0 | 72.5 | 304.4 | -4.2 |
| RUN 1140 DRAIN 30 min TOTAL DRAIN, DATA @60 min | 99.2 | 39.3 | 59.9 | 805.2 | -0.805 | 292.4 | 14.2 | 306.6 | 75.0 | 76.0 | 302.0 | -4.1 |
| RUN 1142 DRAIN 30 min TOTAL DRAIN, DATA @60 min | 99.2 | 40.1 | 59.1 | 805.1 | 0.000 | 291.4 | 14.2 | 305.6 | 76.7 | 74.8 | 301.7 | -3.9 |
| RUN 1146 DRAIN 30 min TOTAL DRAIN, DATA @60 min | 99.2 | 39.3 | 59.9 | 805.0 | -0.805 | 292.9 | 14.1 | 306.9 | 76.7 | 77.0 | 301.7 | -4.2 |
| RUN 1158 DRAIN 30 min TOTAL DRAIN, DATA @60 min | 99.2 | 39.3 | 59.9 | 805.0 | -0.805 | 292.9 | 14.2 | 307.1 | 76.7 | 73.0 | 304.2 | -4.1 |
| RUN 1160 DRAIN 60 min TOTAL DRAIN, DATA @30 min | 99.2 | 39.3 | 59.9 | 804.5 | -0.805 | 294.6 | 14.1 | 308.7 | 76.7 | 76.0 | 304.0 | -4.1 |
| RUN 1172 DRAIN 30 min TOTAL DRAIN, DATA @60 min | 99.2 | 39.3 | 59.9 | 805.3 | -0.805 | 293.4 | 14.3 | 307.6 | 76.7 | 73.7 | 304.3 | -3.8 |
| RUN 1178 DRAIN 30 min TOTAL DRAIN, DATA @60 min | 99.2 | 39.3 | 59.9 | 804.5 | -0.805 | 291.5 | 14.1 | 305.6 | 76.3 | 76.3 | 300.9 | -4.4 |
| RUN 118930 min TOTAL DRAIN, DATA @ 60 min | 99.2 | 39.3 | 59.9 | 804.3 | -0.804 | 291.8 | 14.1 | 305.9 | 75.0 | 75.6 | 301.6 | -4.0 |
| RUN 120130 min TOTAL DRAIN, DATA @ 60 min | 99.2 | 39.3 | 59.9 | 804.3 | -0.804 | 288.1 | 14.2 | 302.3 | 75.0 | 73.7 | 299.0 | -3.9 |
| RUN 120960 min TOTAL DRAIN, DATA @ 30 min | 99.2 | 39.3 | 59.9 | 804.5 | -0.805 | 294.3 | 14.0 | 308.3 | 75.0 | 72.0 | 306.0 | -4.3 |

[^21]Table 25 - Summary of Regular Test Data for Large Tank (41.4 gallon)

| Test Run/Action | Tank Total Volume | Water Volume in Tank | $\mathrm{N}_{2}$ Volume in Tank | Weight <br> Water <br> Drain | Change Water Volume | Tank Pres. | Ambient Pres. | Tank Pres. | Ambient Temp | Tank <br> Temp <br> (gas) |  | Actual <br> $\Delta$ Tank <br> Pressure <br> @NT |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\mathrm{a} / \mathrm{n}$ | L | L | L | g | L | psig | psia | psia | ${ }^{\circ} \mathrm{F}$ | ${ }^{\circ} \mathrm{F}$ | psia | psia |
| RUN 126730 min TOTAL DRAIN, DATA @60 min; Start Pres 299.1 psig | 156.7 | 62.6 | 94.1 | 805.7 | -0.8 | 296.5 | 14.4 | 310.9 | 74.1 | 71.6 | 308.8 | -2.6 |
| RUN 127330 min TOTAL DRAIN, DATA @60 min | 156.7 | 62.6 | 94.1 | 804.7 | -0.8 | $\underline{291.6}$ | 14.1 | 297.2 | 75.0 | 71.5 | 295.3 | -2.8 |
| RUN 127730 min TOTAL DRAIN, DATA @60 min | 156.7 | 62.6 | 94.1 | 805.2 | -0.8 | $\underline{295.0}$ | 14.2 | 309.2 | 75.0 | 72.0 | 306.9 | -2.7 |
| RUN 128030 min TOTAL DRAIN, DATA @60 min | 156.7 | 62.6 | 94.1 | 805.2 | -0.8 | 292.9 | 14.2 | 307.1 | 75.0 | 71.8 | 304.9 | -2.6 |
| RUN 128760 min TOTAL DRAIN, DATA @30 min | 156.7 | 62.6 | 94.1 | 804.5 | -0.8 | $\underline{291.2}$ | 14.2 | 305.4 | 75.0 | 71.3 | 303.5 | -2.6 |
| RUN 130370 min TOTAL DRAIN, DATA @30 min | 156.7 | 62.6 | 94.1 | 804.5 | -0.8 | $\underline{291.3}$ | 14.2 | 305.5 | 75.0 | 78.7 | 299.5 | -2.6 |

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[^0]:    ${ }^{1}$ Petition for Rulemakings - FMVSS 303 and Liquid Propane, , Date Jan. 3, 2011. Docket No. NHTSA-2011-0002-0001
    ${ }^{2}$ LPG is commonly known as propane. Propane typically consists of $\mathrm{C}_{3} \mathrm{H}_{8}$ (majority as propane) and $\mathrm{C}_{4} \mathrm{H}_{10}$ (minority as butane). For our calculations we assumed liquid is all propane. Ref: www.afdc.energy.gov/fuels/fuel properties.php (Retrieved October 9, 2014)
    ${ }^{3}$ The physical state of propane is a pressured liquid. At normal atmosphere pressure and temperature above $-44^{\circ} \mathrm{F}$ propane remains in its gas form. At lower temperature and/or higher pressure, propane will become a liquid.
    ${ }_{5}^{4}$ Propane is stored under pressure in the tank. As pressure is reduced/released the liquid propane is vaporizes and turns into gas.
    ${ }^{5}$ Note: propane is typically delivered to the vehicle engine as a liquid. It goes into a reducer (European term). a device through which the liquid propane is converted to gaseous phase, reducing and adjusting its pressure. The reducer receives heated water from the engine cooling system.

[^1]:    ${ }^{6} \mathrm{R}=8.3144598(48)(\mathrm{J} /$ mole * K) in SI units http://physics.nist.gov/cgi-bin/cuu/Value?r|search_for=gas+constant (retrieved June 28, 2016) or $8.3144598(48)(\mathrm{L} * \mathrm{kPa}) /(\mathrm{mole} * \mathrm{~K})$. Standard uncertainty is $0.0000048(\mathrm{~J} / \mathrm{mol} * \mathrm{~K})$
    ${ }^{7}$ Based on comparing ideal gas calculations to density calculations in our range of interest from Thermophysical Properties of Fluid Systems" in NIST Chemistry Web Book for temperature and pressure We also used (1/1) or one/one to specify a dimensionless value, and ( $\mathrm{a} / \mathrm{n}$ ) for an alphanumeric value.

[^2]:    ${ }^{8}$ For filling the tanks, all measurements of water were done by weight so the compressibility of water for initial tank fill and drainage was not considered. Also, compressibility of water was not evaluated in any other calculations. Actual density of water at $68^{\circ} \mathrm{F}, 14.696 \mathrm{psia}$ is 998.20715 $\mathrm{kg} / \mathrm{m}^{\wedge} 3$ and at $68^{\circ} \mathrm{F}, 310.0 \mathrm{psia}$ is $999.13784 \mathrm{~kg} / \mathrm{m}^{\wedge} 3$. E. W. Lemmon, M. O. McLinden \& D. G. Friend, "Thermophysical Properties of Fluid Systems" in NIST Chemistry WebBook, NIST Standard Reference Database Number 69, Eds. P. J. Linstrom and W. G. Mallard, National Institute of Standards and Technology, Gaithersburg MD, 20899, http://webbook.nist.gov, (retrieved April 1, 2015).
    ${ }^{9}$ A standard requirement is that LPG tanks be only filled to $80 \%$ volume to allow for liquid expansion. For testing we used water, which is approximately twice as heavy as LPG. Then $40 \%$ water will allow the tank system to be at a weight that a test vehicle would see.
    ${ }^{10}$ See Appendix C1 - Calculations LPG Equivalent and Energy Content Values
    ${ }^{11}$ Actual shut off of water may vary the amount slightly from the target amount of 805 grams. Actual flow rates will also vary due to changing flow rates as the tank pressure decreases.
    12 The figure of 805 g is total allowable from FMVSS 301.
    ${ }^{13}$ In FMVSS No. 303, the maximum allowable leak of CNG is the greater of 154 psi and $895 * \mathrm{~T} / \mathrm{V}_{\mathrm{F} \text { kiloPascals. }}$. The 154 psi corresponds to $5 \%$ pressure drop from a 3,000 psi tank, which was considered as the cumulative error in pressure measurement. See April 25, 1994 Final rule establishing FMVSS No. 303. [59FR19648]
    ${ }^{14}$ Since 154 psi corresponds to $5 \%$ error in pressure measurement for a 3,000 psi tank, $5 \%$ error in measurement for a 300 psi tank (typical LPG tank) is 15.5 psi .

[^3]:    ${ }^{15}$ All calculations and pressure measurement values are performed using English units and are specified in gauge pressure (psig) unless otherwise noted. For our tanks maximum operating pressure is 312 psig so typical starting test pressure would be 20 psig lower.
    ${ }^{16}$ The precise test start operating pressure reduction is approximate since it requires multiple pressurizations and incremental pressurizations to achieve the approximate starting operating pressure due to cooling of the tank.
    17 stabilized at ambient temperature (or other defined stabilization condition)

[^4]:    ${ }^{18}$ STP as commonly used in the Imperial and USA system of units - as air at $60^{\circ} \mathrm{F}\left(520{ }^{\circ} \mathrm{R}\right)$ and $14.696 \mathrm{psia}\left(15.6^{\circ} \mathrm{C}, 1 \mathrm{~atm}\right)$; For these calculations, we used what is technically, NTP or Normal Temperature and Pressure, defined as air at $20^{\circ} \mathrm{C}\left(293.15 \mathrm{~K}, 68^{\circ} \mathrm{F}\right)$ and 1 atm $(101.325 \mathrm{kN} / \mathrm{m} 2,101.325 \mathrm{kPa}, 14.7 \mathrm{psia}, 0 \mathrm{psig}, 30 \mathrm{in} \mathrm{Hg}, 760$ torr $)$. Also note that STP - Standard Temperature and Pressure - is defined by IUPAC (International Union of Pure and Applied Chemistry) as air at $0^{\circ} \mathrm{C}\left(273.15 \mathrm{~K}, 32{ }^{\circ} \mathrm{F}\right)$ and $10^{5} \mathrm{Pascal}$. www.engineeringtoolbox.com/stp-standard-ntp-normal-air-d_772.html (Retrieved October 9, 2014)

[^5]:    ${ }^{19}$ See Appendix A3 - Equipment and Instrumentation, for detailed information on components and instrumentation.
    ${ }^{20}$ The test tanks are automotive fuel tanks currently in production from Manchester Tank Corporation, 1000 Corporate Centre Drive, Suite 300 , Franklin, TN 37067, Autogas tanks, liquid, vapor or dual service options, $80 \%$ stop fill device, seal welded stainless steel, with ASME nameplate; 312 PSI maximum allowable working pressure, and meet AASME Section VIII, Division I Code and meet NFPA \# 58 standards for engine fuel applications.
    ${ }^{21}$ As labeled by the tank manufacturer. See Appendix D - Tank Technical Data, for detailed test tank specifications, detailed drawings, serialization specifications and other information.

[^6]:    ${ }^{22}$ Signal conditioning of the data was provided by Analog Devices 3B series signal conditioning. Data was converted from analog to digital using a Measurement Computing Inc. PCMCIA DAS16/16 card, controlled by VRTC written software DAS16 version V14B. Data was acquired at a rate of 200 samples per second. Data was then averaged over a 100 sample interval and then subsampled to 2 samples per second. Post processing of the data was accomplished by using VRTC written software program DSAN_Win, version: 14C, version date: Oct-03-2014. Post processing consisted of filtering using a Butterworth 2-pole low pass phaseless filter with a 0.1 Hz cutoff ( -3 dB ) frequency.
    ${ }^{23}$ It was decided to run tests in a bay where some temperature changes occur so that it could more realistically evaluate conditions that would be experienced during an actual test. Additionally, ambient air pressure (raw) was evaluated so that gauge pressure readings could be evaluated as absolute pressures as well. Also, the Transportation Research Center Inc., on which VRTC is located, publishes raw ambient pressure readings as well as temperature, humidity and other readings which can assist in verifying any atmospheric conditions which might affect the tests.

[^7]:    ${ }^{24}$ See Appendix D — Detailed Tank Specifications for overall tank specifications for both tanks and Tank Technical Data, Table 23 for engineering specifications
    ${ }^{25}$ Expansion principles that affect the liquid propane volume were factored into what would be a typical delivery to an end use tank (the example here is a tank used for heating). Assume for instance, that a 250 gallon propane tank has 100 gallons of propane at $6^{\circ} \mathrm{F}$. The industry standard $60^{\circ} \mathrm{F}$ is universally recognized as the base reference point for liquid propane volume correction. A properly functioning float gauge will read $40 \%$. Many propane delivery trucks and vehicle propane filling stations have meters that measure the amount of propane pumped into consumer tanks. These meters include a volume correction device known as an automatic temperature compensator. The temperature compensator takes into account the temperature of the liquid propane running through the meter and automatically adjusts to correctly deliver the amount of propane requested. www.propane101.com/propanevolumecorrection.htm (Retrieved December 12, 2014)

[^8]:    ${ }^{26}$ Since the tank maximum working pressure (MAWP) was specified as gauge pressure and the CNG tank test specifications used gauge pressure, we monitored tank pressure in psig. This would be a more conservative value when compared to absolute pressure. Of course, air pressure (raw) was also monitored to evaluate the effects that using an absolute pressure gauge would have on the outcome of the testing.

[^9]:    ${ }^{27}$ Raw refers to ambient air pressure (barometric pressure) at a given altitude not corrected to mean sea level.

[^10]:    ${ }^{28}$ See Appendix E - Other Data Runs, for detailed test tank of regular leak rate results on the small tank (corrected for ambient pressure change and tank temperature). Additional runs for practice and for fast and slow leak rates were also made.
    ${ }^{29}$ The target total drain was 805 g water in 30 minutes. For testing, the rate was calculated based on a predetermined time for each 100 g of water ( 100 g in 212 seconds). The cumulative and interval time were then compared to this precalculated time and fine adjustments were made as needed. No attempt to vary the initial allowed loss" first 5-minute amounts was made. Total time may be slightly higher than 30 minutes.
    ${ }^{30}$ The target amount to drain in the GM Petition presentation was target of 804.2 g . We rounded up for clarity. Also note that there was a typical overrun of water drained. For this run, the total of 805.2 g water was drained in 29 minutes, 46.9 sec .
    ${ }^{31}$ Under the column "Run No.", " 30 min DRAIN @ 30 min" means we drained water for 30 minutes and read all data at 30 minutes. Also " 30 min DRAIN @ 60 min" means we drained for a total of 30 minutes and took data at 60 minutes allowing for temperature and pressure stabilization time.

[^11]:    ${ }^{32}$ Note that slight variations in "leak" rate occurred due to fine adjustment of needle valve controlling flow and changes in leak rate as pressure diminished in tank.
    This is also evident in the initial leak rate as shown in the graph. Actual target rate was 805 g in 30 minutes.
    ${ }^{33}$ For all graphs blue represents pressure and red is temperature. Drain time for all tests start at 100 seconds.

[^12]:    ${ }^{34}$ See Appendix C2 - Gaseous Leak Rates, for details.

[^13]:    ${ }^{35}$ See Appendix E - Other Data Runs, for detailed test tank of regular leak rate results on the large tank (corrected for ambient pressure change and tank temperature). Additional runs for practice and for fast and slow leak rates were also made.

[^14]:    ${ }^{36}$ The Canadian standard of either/or a $5 \%$ drop in pressure temperature corrected pressure or 142 g of fuel loss. Reference Transport Canada, Standards and Regulations Division, Test Method 301.1, LPG Fuel System Integrity, Revised: February 8, 2001, with update to SOR/2008, 104 April 16, 2008, 301.1 Section (2) (a) and (b).
    ${ }^{37}$ Transducers are available with up to $+/-0.05 \%$ FS. Also consider a 5 volt or 10 volt full scale output in lieu of mv/V output.

[^15]:    ${ }^{38}$ If leakage rate is used in actual testing, since flow out of the tank will not stop after completion of the test, recovery estimates would have to be made to evaluate the true pressure drop to evaluate total energy loss.
    ${ }^{39}$ For Figure 16 and 17, and graphs of temperature and pressure, red $=$ temperature in degrees Fahrenheit and blue $=$ pressure in psig.

[^16]:    ${ }^{40}$ For example, the MAWP (or as specified by the tank manufacturer, the service pressure in psig), is 312 psig so 20 psig less would be 292 psig. Correcting for tank temperature (see appendix for allowable tank pressures), and using the maximum and minimum temperature range as specified by the CNG test as well as $+/-2 \%$ accuracy gives us the desired pressure to be used at test initiation.
    ${ }^{41}$ It should be noted, that unlike CNG testing, with initiating pressure of 3000 psig or higher, we are only using 300 psig, so the temperature and pressure changes during the filling operation are not as high.

[^17]:    ${ }^{42}$ We are not trying to develop a heat transfer function, but merely want to show change in tank temperature in a typical test environment.

[^18]:    ${ }^{43}$ Numbers do not necessarily represent the uncertainty of a value based on the number of significant figures but rather the number of active digits is presented for understanding of the measurement or change in the figure of merit being used.
    ${ }_{45}$ Those would be similar to changing to absolute pressure and then correcting to normal temperature.
    ${ }^{45}$ The intent of this analysis was to evaluate a pressure drop that according to some calculations would be required to correlate to a verifiable measured amount. This amount would have been the minimum amount according to a value based on the accuracy of a pressure measurement that has been shown in other presentations to be unreasonably large (e.g. using sum instead of RSS cancelations to aggregate uncertainties, etc.).

[^19]:    ${ }^{46}$ We also assume a compressibility factor for nitrogen, Z of unity. Actual compressibility factor based on "Thermophysical Properties of Fluid Systems" in NIST Chemistry WebBook is approximately 0.9976 to 0.9979 .
    ${ }^{47}$ We merely wish to illustrate a variation in leak rate if gas is escaping instead of a liquid. We are of course not taking into account the fact that with propane (instead of nitrogen), as the pressure in the tank would decrease additional propane would go from a liquid phase to a gaseous phase thus increasing the pressure to some extent. Also as liquid propane comes out it could flash and freeze and that could change the pressures and the amount of available fuel. Additionally, the issue with using nitrogen for propane is the viscosities of the two gases are different. For the same pressure of nitrogen gas and propane the velocity of nitrogen is about 1.5 times faster. Pressure would remain higher if more propane entered the gaseous phase increasing loss, while freezing and taking into account different velocities lower the actual ratio.

[^20]:    ${ }^{48}$ Labels on the tanks as received are as follows: $\mathrm{SG}=$ sight gauge. A float gauge indicates the liquid level inside the tank. V6=Fill valve with an overfill prevention device. V4= fixed maximum liquid level gauge. V3= vapor opening. Plugged: 3/4 NPT. V2= relief valve with a 312 psi start to discharge setting. V1= service valve opening with 12 volt solenoid operated valve with a $3 / 8$ " SAE outlet. The labels under V6 indicate the function of the fill valve in both English and French. The service openings are labeled as to their communication with either the vapor or liquid space of the container in both English and French.
    ${ }^{49}$ Note the "weight" of propane is an approximation. The density of propane is actually a function of temperature since it can exist in a liquid/gas phase at temperatures over the boiling point. Liquid density propane ( 1.013 bar at boiling point) : $580.88 \mathrm{~kg} / \mathrm{m} 3 \mathrm{Liquid} /$ gas equivalent $\left(1.013 \mathrm{bar}\right.$ and $15^{\circ} \mathrm{C}\left(59{ }^{\circ} \mathrm{F}\right)$ )
    Ref: http://encyclopedia.airliquide.com/Encyclopedia.asp?GasID=53\#LiquidGasConversion (Retrieved March 22,2016 revised)
    ${ }^{50}$ For this chart and tests for consistency we used $4.22 \mathrm{lbm} /$ gal as presented in the chart Converting Gasoline Energy to Equivalent CNG/LPG Pressure Drop from "GM Petition, FMVSS No. 303 Upgrade, NHTSA, MRT Briefing," September 18, 2012. This gives a value of $505.6 \mathrm{~kg} / \mathrm{m}^{\wedge} 3$.
    ${ }^{51}$ Note that Industry Canada/Measurements Canada specifies $510 \mathrm{~kg} / \mathrm{m}^{\wedge} 3 \mathrm{C}_{3} \mathrm{H}_{8} @ 15^{\circ} \mathrm{C}$ (or $4.256 \mathrm{lbm} / \mathrm{gal} \mathrm{C}_{3} \mathrm{H}_{8} @ 15{ }^{\circ} \mathrm{C}$ or $\sim 60{ }^{\circ} \mathrm{F}$ ). Measurements Canada $\underline{\mathrm{https}: / / \mathrm{www} . i c . g c . c a / e i c / s i t e / m c-m c . n s f / e n g / l m 00136 . h t m l}$ (Retrieved 01/05/2014:1018). We used the $4.22 \mathrm{lbm} / \mathrm{gal}$ value for our calculation of weight and volume. However all temperature and pressure calculations were corrected to NIST NTP as previously noted.
    ${ }^{52}$ Also "the industry standard $60^{\circ} \mathrm{F}$ is universally recognized as the base reference point for liquid propane volume correction." $\frac{\text { www.propane101.com/propanevolumecorrection.htm (Retrieved January 5, 2015) }}{53}$
    53 Reference Guide For Stock Autogas Tanks, www.mantank.com/pdf/AutogasTankGuide.pdf (Retrieved March 15, 2015)

[^21]:    54 "RUN1178 DRAIN 30 min TOTAL DRAIN, DATA @ 60 min" means we drained water for 30 minutes and then let conditions stabilize and read all data at 60 minutes. "RUN 1125 DRAIN 60 min TOTAL DRAIN, DATA @30 min" means we drained for a total of 60 minutes and took data at 30 minutes, which means that there was no stabilization time. Other runs were tests to verify set up and fast and slow drain rates.

